

### PETRO-CANADA OIL AND GAS

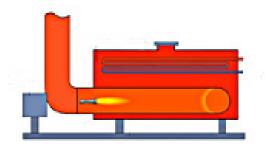
# FIRE-TUBE IMMERSION HEATER OPTIMIZATION PROGRAM & Field Heater Audit Program

### Energy Management Work Shop "The Fuel Gas Challenge"



by **Phil Croteau** P. Eng. Energy Efficiency Engineer

January 15-17, 2007





## Outline (25 min session)

- Overview Top 5 Priorities (PTAC TEREE)
- PTAC TEREE: the Origin of the "Fire-tube Heater Study"
- Combustion Efficiency. Excess Air
- Heat Transfer Fire-tube Design
- Combustion Efficiency Fire-tube Selection
- Combustion Efficiency Heat Flux Rate
- Burner Selection
- Burner Duty Cycle
- Combustion Efficiency Reliability Guidelines
- Heater Tune-up Inspection Procedure
- Insulation
- PCOG Fire-tube Immersion Heater Optimization Program
- Field Audit Program (NRCAN Energy Audit Incentive Program)
- Conclusion, Q&A



### Overview: Top 5 Priorities, ER & EE



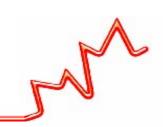
### PTAC - TEREE Study 2004-2005: Top 5 Priorities for ER and EE

Petroleum Technology Alliance Canada

- Technology for Emission Reduction and Eco-Efficiency
- 1. Venting of Methane Emissions
- 2. Fuel Consumption in Reciprocating Engines
- 3. Fuel Consumption in Fired Heaters
- 4. Flaring and Incineration
- 5. Fugitive Emissions



### **Background**



- Common concern for many upstream operating companies is the energy consumption associated with immersion heaters
- Energy often used to fire these heaters is high quality refined sales gas
- Common problem with the immersion heaters is that they may have low fuel efficiencies between 30% and 60%
- Compared to common boiler technology these heaters should be able to run at 70 to 80% efficiency
- Recent estimates suggest that heaters currently waste in excess of 2 to 3 billion BTU/hr of fuel (1360 to 2040 e3m3/d gas) that could be conserved to generate added sales
- At an average cost of \$5/GJ this represents \$100 to \$150 million of lost revenues due to inefficient use of fuel gas
- Also represents an associated 1.5 million additional tonnes of carbon dioxide being discharged into the atmosphere per year



### **Project Sponsors**



The following sponsors collaborated with PTAC to provide financial and technical support:

Petro-Canada Shell Canada

EnCana Corporation CETAC-West

Husky Energy CAPP

Nexen BP Canada

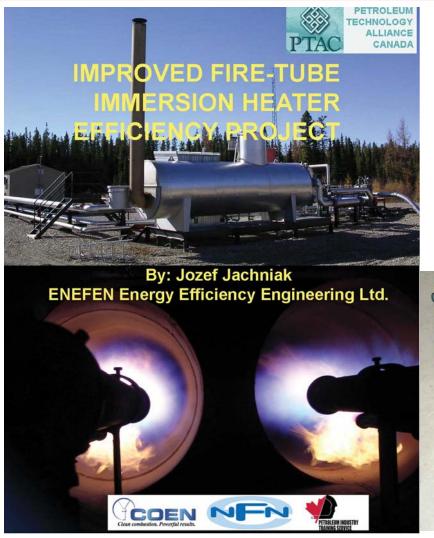
The report has now been publicly released, and is available to view on the PTAC website at

www.ptac.org/techeetteree.html





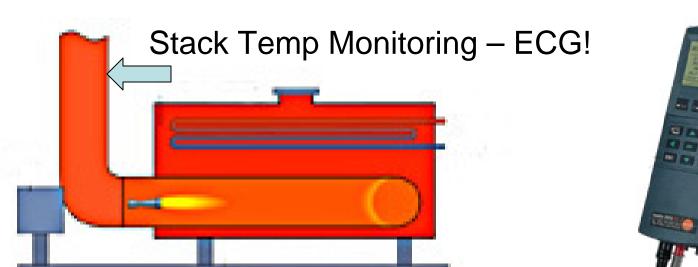








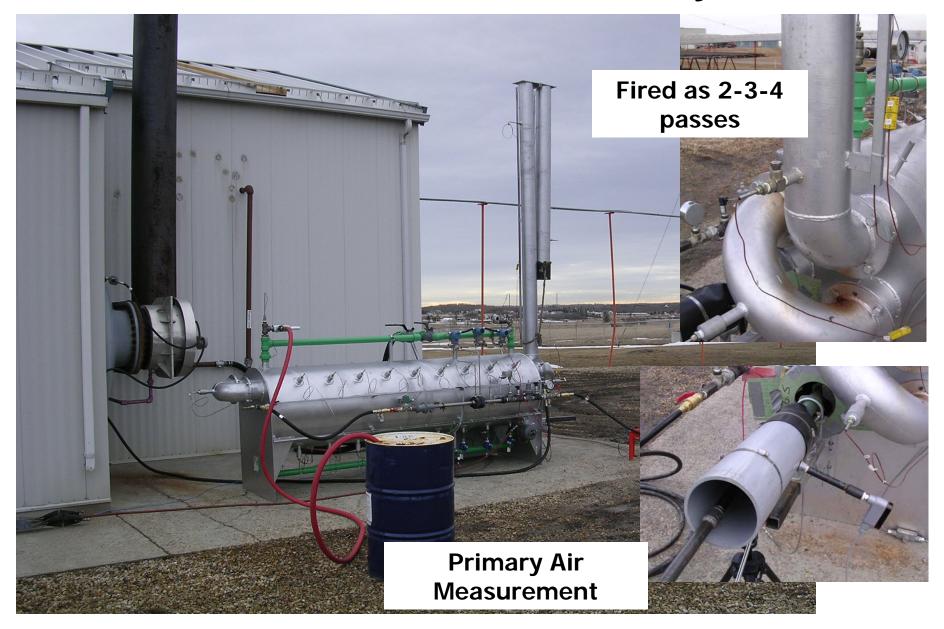
### - We built a heater, fired it with several different burners!







### **PTAC Lineheater Study**



### **Burner Vendors Participating**

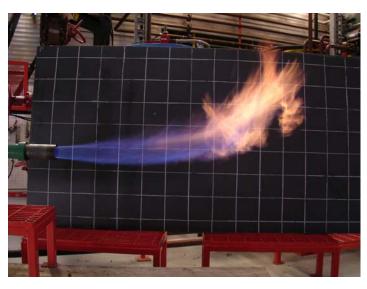
- → A-Fire
- → ACL
- → Bekaert (MCI) (3) 7 combinations
- → Eclipse
- → Hauck
- → Kenilworth (4)
- → Maxon (3)
- → North American
- → Pro-Fire (2)
- → Pyronics (4)



10 burner vendors = 25 burners tested

### **TESTS – OPEN FLAME TESTS**









### **HEATER TEST STAND - INSTRUMENTATION**

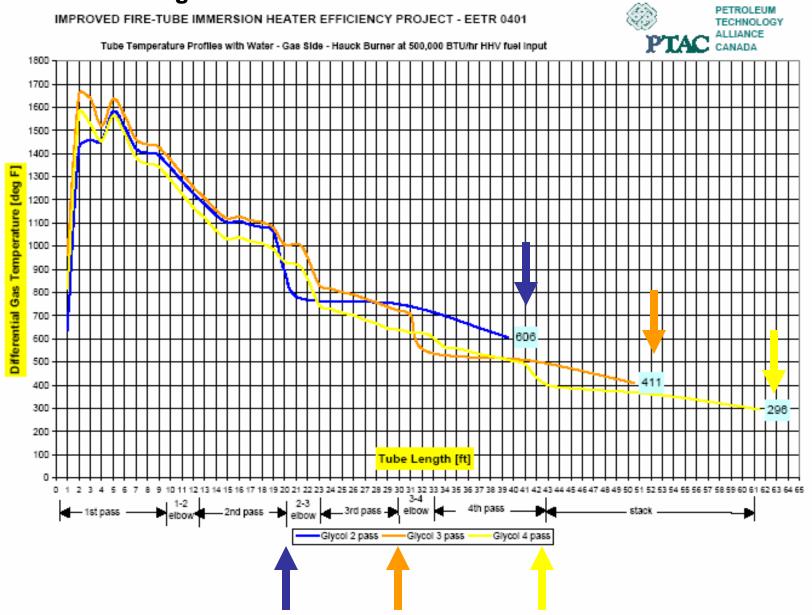


DCS control and data recording

### **Heat Transfer**

### PTAC Test - Glycol, 2-3-4 passes

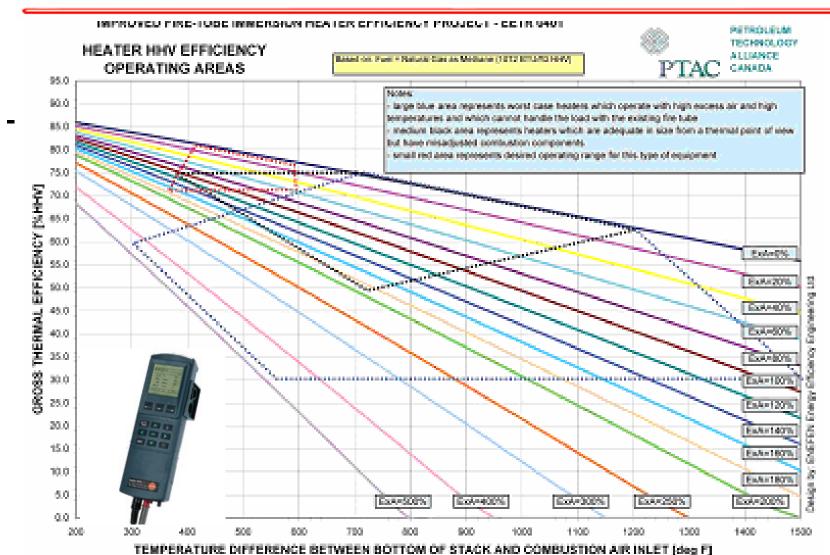
### - Fire-tube Design



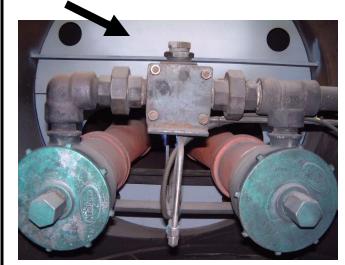


## Combustion Efficiency – Excess Air The GOOD, the BAD & the UGLY!





## Combustion Air Control





Excess air baffles!





As found: fouled flame cell!

Excess air 0.0%

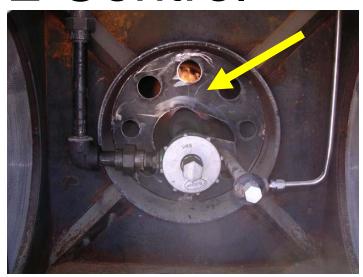
Stack CO >110,000 ppm!

Flame cells are not filters!

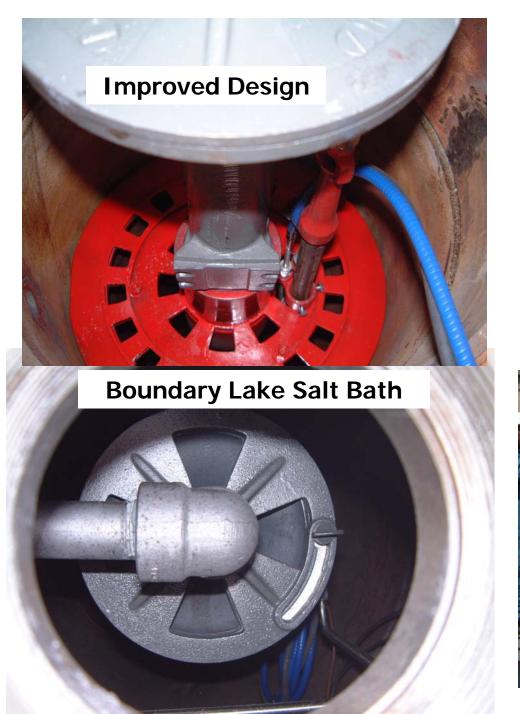
## Excess Air/O2 Control









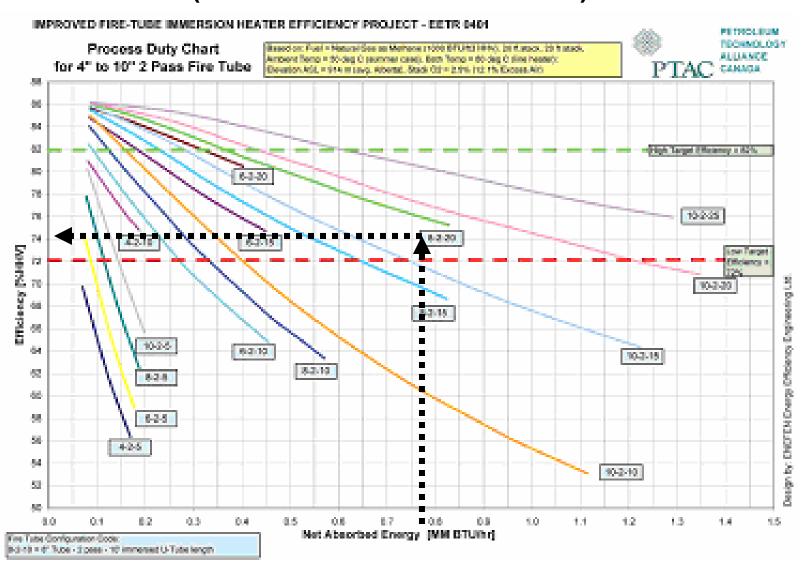




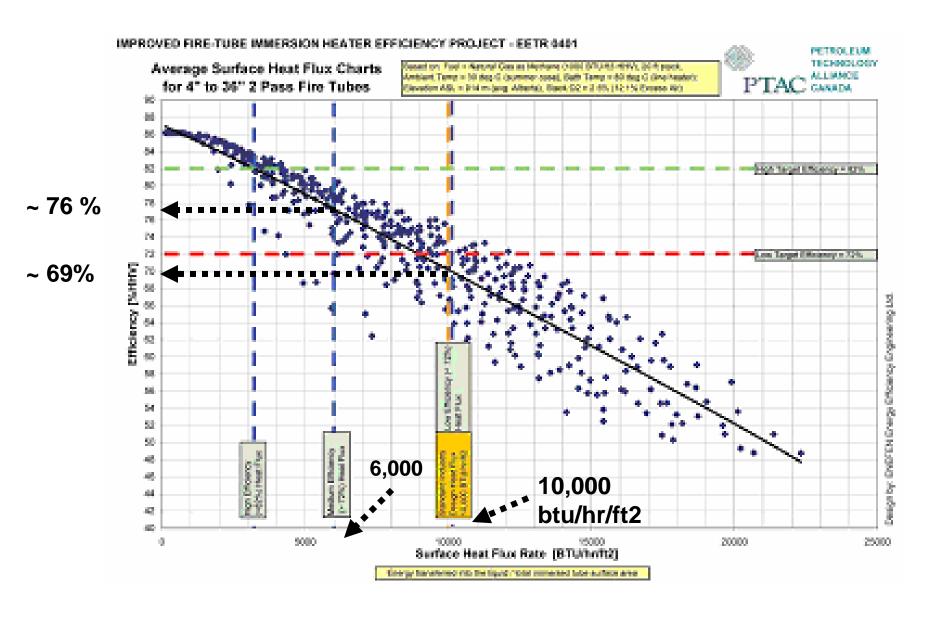


### **COMBUSTION EFFICIENCY**

## - IMPACTED BY FIRE-TUBE SELECTION (SENSIBLE HEAT RECOVERY!)



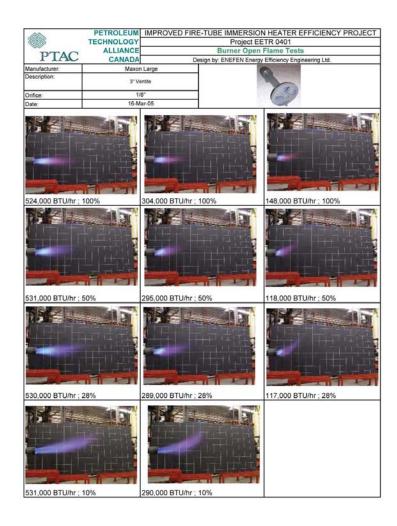
## COMBUSTION EFFICIENCY - IMPACTED BY FIRE-TUBE HEAT FLUX RATE



### **Burner Selection**

## - HIGH PRIMARY AIR INSPIRATION, TURNDOWN, FUNCTIONALITY - MAXON VENTITE

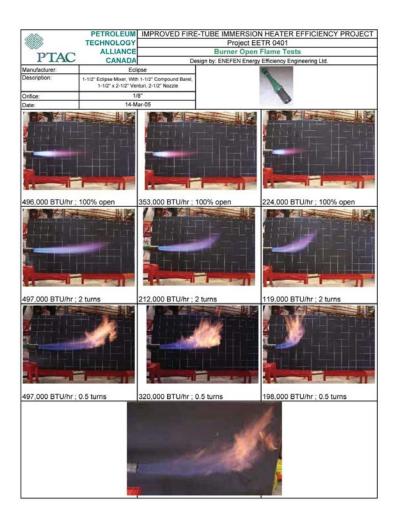
@A	PETROLEUM	IMPROVED F	FIRE-TUBE IMME	ERSION H	EATER EFFICIENCY PROJECT
	TECHNOLOGY		Pro	ject EETR	0401
DTAC	ALLIANCE		Burne	er Charact	eristics
PTAC	CANADA				ciency Engineering Ltd.
Manufacturer:	Maxon	Large	Address:	637	5 Dixie Road, Unit 3
Description:	3" Ver	ntite			
	200000	Mess.	City, Province, C		sissauga, ON L5T 2E1
Orifice:	1/8		Telephone / Fax		5) 795-0717/(905) 795-1819
Overall Length:	21		Web Site:		p://www.maxoncorp.com neral Arrangement
	8			VI	mpact burner assembly features gas nozz venturi, miker, and primary air shutter nbination. Heavy duty cast iron componer
		100 E 113		Gas	s Nozzie
C		0	0	or or	eavy duty cast iron nozzle includes interni lame retention device with large main gas fifee and 8 smaller holes located around it berimeter. Available with integral pilot and flame rod mount (PilotPak).
		6		G st	is Mixer & Primary Air Adjustment  sas mixer features a low entrance loss bel apped inlet. Heavy duty cast iron "register pes shutler includes a locking screw. Gas connection through the back of the mixer, mple rear access to the orfice by unbotlin the back plate of the register.
				Car	condary Air Adjustment
			9	grabyst.	No secondary air adjustment incorporated



### **BURNER SELECTION**

### - ECLIPSE

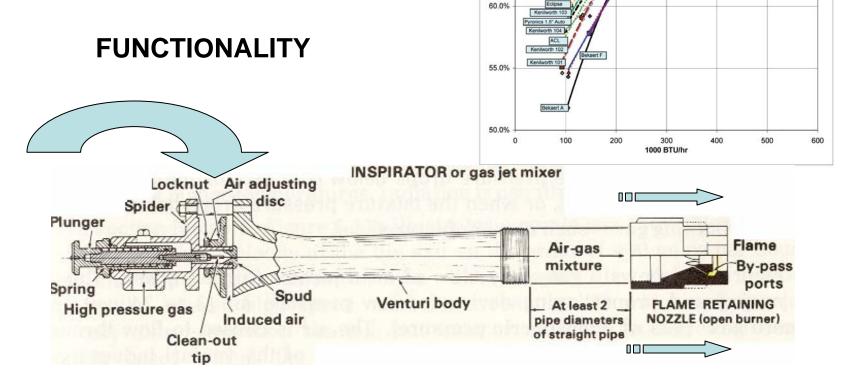
<i>A</i>		IMPROVED FIR	E-TUBE IMMERSIO	ON HEATER EFFICIENCY PROJECT
***	TECHNOLOGY	review and the real of the life		ETR 0401
PTAC	ALLIANCE			aracteristics
	CANADA			gy Efficiency Engineering Ltd.
Manufacturer:	Edi		Address:	#5,3530-11A Street N.E.
Description:	1-1/2" Eclipse Mixer, With 1-1/2" x 2-1/2" Ver	1-1/2" Compound Barrel, hturi, 2-1/2" Nozzle	City, Province, Code:	Calgary, AB T2E 6M7
Orifice:	1/	8*	Telephone / Fax:	(403) 291-9211/(403) 291-9214
Overall Length:	30	0"	Web Site:	www.eclipsenet.com
-			4000	General Arrangement  Typical complete assembly of Eclipse burner common in the industry. Assembly consists a mixer, compound barrel, Venturi, and gar
	3			Gas Nozzle  Eclipse Ferrofix Nozzle with built-in flame retention feature. Nozzle produces long an narrow flame pattern as compared to a widname available with Sticktte nozzles.
				Gas Mixer & Primary Air Adjustment  Eclipse mixer commonly used in the industrals ob ysome of the other burner manufacturers. Basic mixer features cast inc body with gas orfice and primary air shutte Ads supplied with the burner is a needle val which allows fine adjustment to the orfice opening. The optional compound barrel is us to enhance fuellair mixing, and is recommended for heavier fuel gases.
	1	01		Secondary Air Adjustment  No secondary air adjustment incorporated



### **BURNER SELECTION**

- HIGH PRIMARY AIR INSPIRATION,

TURNDOWN,



TECHNOLOGY

PTAC

80.0%

75.0%

70.0%

ALLIANCE

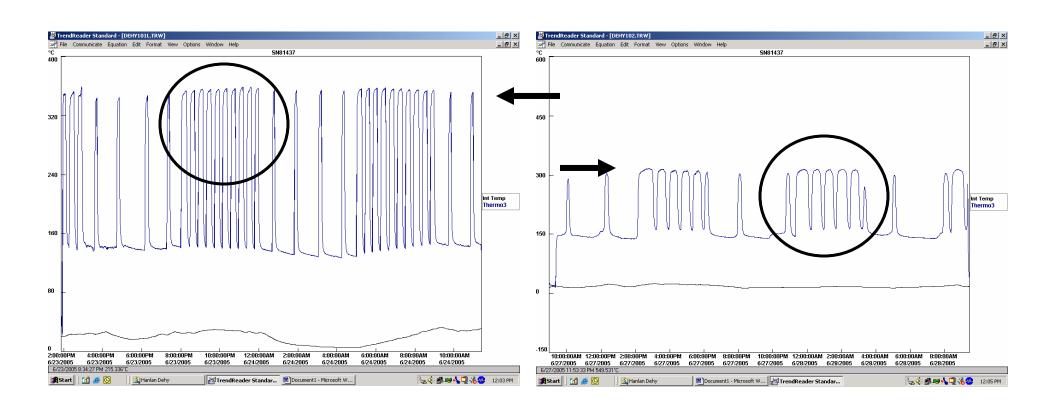
CANADA

Project EETR 0401

THERMAL EFFICIENCY % HHV

### **Burner Duty Cycle Management**

- short duty cycle at high firing rate vs. the longer duty cycle firing at a lower rate



<u>Duty Cycle to the Extreme</u> - This is the consequence of an extremely low main burner duty cycle, only the pilot ran, condensing moisture in "Products of Combustion". Water accumulates and freezes at the flame cell as it tries to drain out. Level rises until even the pilot is extinguished! This is a concern for oversized heaters, more common a problem than we accept.





## Combustion Efficiency, Emission and Reliability Guidelines 4 Pages

### IMMERSION HEATER FIELD INSPECTION AND EFFICIENCY EVALUATION REPORT



PETROLEUM TECHNOLOGY ALLIANCE CANADA

COMBUSTION EFFICIENCY, EMISSIONS AND RELIABILITY GUIDELINES

Design by: ENEFEN Energy Efficiency Engineering Ltd.

EFFICIENCY DEFINITION: Efficiency is defined as the percentage of gross BTU input that is realized as useful BTU output of a 1 heater. There are two ways of calculating this efficiency: the HHV efficiency uses the higher heating value of fuel input, and the LHV efficiency uses lower heating value of fuel input.

LHV AND HHV BASED EFFICIENCY CALCULATIONS For example pure methane HHV = 1012 BTU/cuft and LHV = 911 BTU/cuft, and the difference is the amount of energy used to evaporate water produced during the combustion process from the hydrogen contained in the fuel. Hence for the same combustion process using methane as a fuel the LHV efficiency value is about 10% higher than the HHV efficiency value. Where the LHV efficiency is easier to use for evaluation of traditional style heaters which do not condense water out of the products of combustion, it cannot be meaningfully used for newer condensing type heaters. In addition, since fuel is measured and sold based on its HHV value, only the HHV based efficiency should be used for the economic evaluation of the heater performance. LHV based efficiency is typically used in the US and the HHV efficiency is more commonly used in Canada. All regulatory requirements in Canada related to burner and fuel controls rating are based on HHV of fuel. Since many heater specifications and many combustion analyzers do not clearly state the basis for efficiency calculations, caution should be exercised when using these efficiency values.

COMBUSTION EFFICIENCY - OVERALL COMBUSTION EFFICIENCY - FUEL EFFCIENCY - HEATER THERMAL EFFICIENCY. These terms are used in the industry interchangeably, although with a fair amount of conduston. To clarify: any of these efficiency terms is based on the calculation of 100% of energy input into the heater (expressed in either LHV or HHV terms) minus the summation of all the losses from that heater, which equals to the useful heat output to the process load. The losses can be either combustion celated or, heater despin specific.

#### COMBUSTION LOSSES FROM THE HEATER: These losses include:

- latent heat of evaporation to moisture in the stack formed from oxidation of hydrogen in the fuel
   unburned fuel (VOC's in the stack) including hydrocarbons, CO, soot (free carbon), H2S or any other combustible compound which did not get oxidized to form CO2 or H2O
- sensible heat lost to heat the product of combustion above the ambient air temperature. Products of combustion include also nitrogen, excess oxygen and HZO vapour from ambient air humidity and possibly the unburned fuel which do not take part in the combustion process but are also heated to the stack temperature. Note that besides combustion air, ambient tramp air can also infiltrate the heater through cracks and openings, however that tramp air would be then included in the products of combustion.

#### HEATER DESIGN SPECIFIC LOSSES: These losses include:

- wall / piping / insulation losses the energy which radiates out of the heater into the surroundings and is carried away by air (wind), foundations, or connecting equipment. Note that only the heat loss from the portion of the stack surface below the location of the thermocouple used for the efficiency measurement would be considered as a loss for this calculation.
- opening losses include any products of combustion leaks from the heater other than stack gas.
- 5 conveyor losses include heat carried away by any form of process "conveyor" which does not stay in the process "product". This would also include heat loss through the piping connecting process to the heater.
  - heat storage losses the energy which is stored in the heater steel, insulation, heat transfer medium, connected equipment, foundation, etc. For heaters, which operate continuously the amount of stored energy remains constant after the initial heatup. For heaters which operate in batch mode or which cycle on/off, the amount of stored heat changes and must be replenished every time the heater is refilled and restarted.
- STACK OXYGEN: Stack oxygen level should be maintained between 2% and 4%, which corresponds to between 9.5% and 21.1% excess air. Below 2% oxygen, sharp increase in CO emissions is expected, above 4% oxygen additional excess air "taking a free ride" through the heater decreases the combustion efficiency
- STACK CO: Stack CO levels should be maintained below 400 ppm safety ceiling. Ideally, in a properly tuned system CO levels below 100 are desirable. Typically, depending on the burner design, CO readings increase at low (below 2%) or high (above 11%) oxygen levels. High CO readings indicate incomplete combustion due to insufficient air flow or due to flame quenching with too much air.
- STACK NOx: Stack NOx levels are a function of burner design and specifically flame shape and temperature. Smaller and hotter flames tend to produce higher NOx levels. Also burners with a single fuel Injection port tend to produce higher NOx levels than those with multiple smaller and spread out ports (fuel staging effect). Typically, properly designed natural draft burners produce between 60 ppm and 80 ppm corrected to 3% oxygen (V/V dry basis) in the stack. Within a given burner design NOx formation is pretty well fixed and cannot be changed by regular tune up techniques.

## Heater Tune-up / Inspection Procedure 2 Pages

### IMMERSION HEATER FIELD INSPECTION AND EFFICIENCY EVALUATION REPORT



PETROLEUM TECHNOLOGY ALLIANCE CANADA

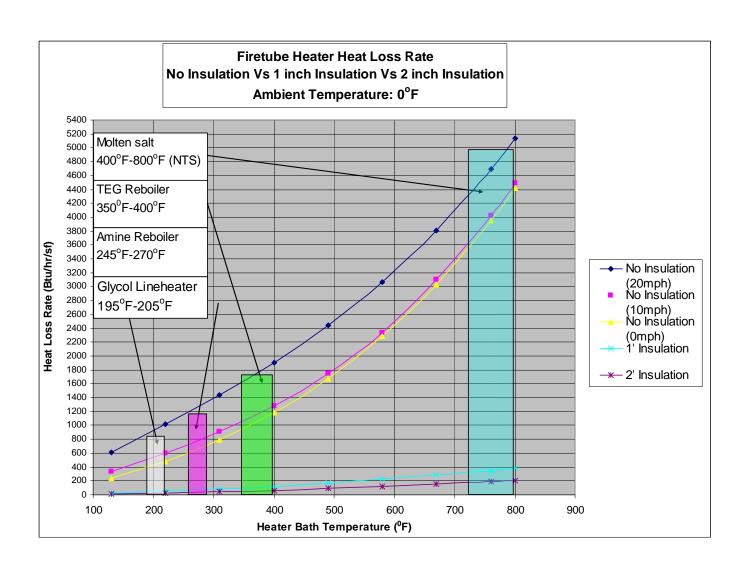
HEATER TUNE UP / INSPECTION PROCEDURE

Design by: ENEFEN Energy Efficiency Engineering Ltd.

- Gas Leak Test: Check area around the heater and inside the fuel train or control enclosure (if present) for safe H2S, O2 and LEL levels
- Visual Inspection: Inspect heater for obvious signs of deterioration, corrosion, damage to instrumentation or fuel train components
- Pilot observation: Check if heater main burner is firing. If not, check if the pilot is on. If pilot is not on check if the fuel to the pilot is turned on and if so turn the pilot fuel off and wait a few minutes for the fire tube to ventilate. Check with
   the control room if there is any reason why the heater is turned off. Once safe to do so, turn the main burner manual fuel valve off, relight the pilot and observe. The pilot should be at least 4" to 8" in length, if smaller, try to increase fuel flow to pilot until "solid" pilot is established.
- Record Heater Data: Record heater data such as make, model, year built, serial number, design process duty, burner 4 type and size, burner orifice size, fire tube OD and length, stack OD and height, etc. as per enclosed inspection and evaluation report sheet.
- Main flame observation: Check all heater permissives such as liquid level LO, temperature HI shutdown, bath temperature setpoint. If everything is oK, open the main burner manual valve. Observe main flame shape, colour, stability, anchoring, noise, impingement on the tube surface.
- Fuel Pressure Measurement: Measure and record fuel gas supply pressure, and main burner pressure (after regulation) while it is firing.
- Fuel Flow Measurement: If available measure fuel gas flow to the main burner by timing the gas meter or measuring pressure drop across the fuel metering orifice. Another simple method is the measurement of the burner gas orifice size and calculation of the gas flow using orifice pressure drop charts. Since the mixture pressure inside the burner Venturi is typically negligible compared to the burner inlet pressure, the burner inlet pressure can be used as an approximation of the pressure drop in the charts. Note, that this method cannot be utilized if the fuel gas orifice is used in conjunction with an adjusting needle valve as it is often the case with Eclipse mixers.
- 8 Heater bath temperature check: Locate bath temperature gauge and record bath temperature. Record also the temperature control setpoint of the temperature controller.
- 9 Stack Measurements: Locate sampling port in the straight length of stack above the fire tube exit from the heater. If no port is available, drill and tap 3/6"UNC hole in the stack. Using combustion analyzer take reading of. Flue Temperature, O2, CO, NOx and efficiency. Record also the ambient air temperature. After taking the sample install a 3/6" bolt using high temperature anti-seize compound or a brass bolt.
- HI CO / LOW O2 with air passages closed: If CO reading is high (in thousands of ppm) and O2 reading very low (close to zero), the heater is being fired substoichiometrically without sufficient oxygen. Remove sample probe from the stack immediately to prevent damaging the CO analyzer cell. Let analyzer purge the cell until CO reading drops to zero. Open access port in the flame arrestor to allow more air flow. Insert analyzer probe back into the stack and observe CO readings. If readings have improved, with the access port open, there is a good possibility that the flame cell is plugged up and needs cleaning. Check also position of any secondary air control devices to make sure that they are not blocking the air flow into the burner.
- HI CO / LOW O2 with air passages open. Burner primary air is misadjusted and must be opened. Open slowly watching the analyzer CO readings until CO levels are low. If there is no or slow reaction, reduce fuel gas pressure to main burner gradually also watching for changes in CO. On some burner models (Eclipse) there could be also a fuel needle valve present which could be adjusted. Note that overfiring of the heater without sufficient combustion air does not increase the heat transfer and it may even decrease it through tube sooting or decrease in the flame temperature. I is also unsafe and may lead to a premature heater failure.
- 12 HI CO / HIGH O2 The indication is that there is too much combustion air. Reduce the primary and secondary air down to 3 to 4% oxygen in the stack.

### **Insulation Heat Loss from Vessel Shell**

- reduction in lost heat (demand) is a 100% saving, adjustments to appliance efficiency, etc. is only partial





## PCOG Fire-tube Immersion Heater Optimization Program



#### PETRO-CANADA OIL AND GAS



#### FIRE-TUBE IMMERSION HEATER OPTIMIZATION PROGRAM

"When you can measure what you are speaking about, and express it in numbers; you know something about it, when you cannot express it in numbers, your knowledge is of a meager and unsatisfactory kind."

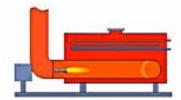
Lord Kelvin (1891)

"if you cannot measure it, you cannot improve it." Lord Kelvin (1895)

"You cannot manage what you do not measure."

Commonly used today!





"You cannot manage what you do not measure."

Philip J. Croteau - P.Eng. Gerald Hewitt - Operations Harley Siebold - Operations Rev. Mar 27, 2006 DRAFT



## Essential Elements of a Heater Optimization Program

### **Executive Summary**

- quantify your number of heaters
- identify/understand their service
- quantify how much fuel they are thought to consume
- make assumptions of their current efficiency
- identify the potential efficiency target and savings
- identify how to get there

### **Statement of Commitment**

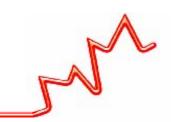
Body of the Program Document

### Conclusions

\* TAKING ACTION TO IMPROVE!



### **Overview: Fire-tube Heater Survey**



### Just how many fire-tube heaters do we/you fire!

- Following is an ~ count of both PCOG and third party. If we don't steward the third party heaters, who will.
- Do we/you have heaters operated by third party?

```
FR – Reboilers: Amine, Glycol ...

FL – Lineheaters: Glycol, Salt Bath ...

FT - Treaters

716
```

Target is to audit 1/3 of heaters per year on 3 yr rotation.



## **PCOG Statement of Commitment**



- excerpt from "Fire-tube Immersion Heater Optimization Program"

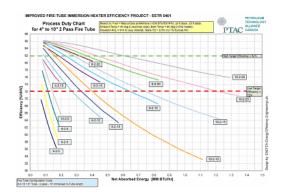
### **Statement of Commitment:**

Through our TLM program, Petro-Canada focuses on improvements in the elements of safety, environment, reliability, economics and the general management of our facilities.

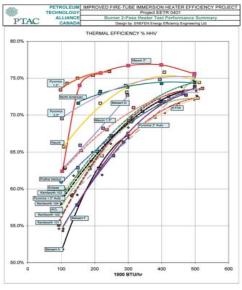
As one of the areas of focus, Petro-Canada had recently committed resources and funding to participate in a study to review and improve our understanding in the design and operation of fire-tube immersion heaters and follow-up with implementation to optimize that equipment. Management is committed to improving the performance of these heaters through expectations of support from Operations, Maintenance and Engineering (OME).

### **Heat Transfer**

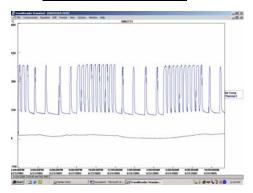
- Fire-tube Design



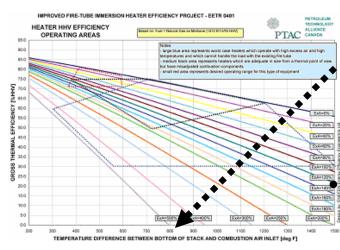
### **Burner Selection**

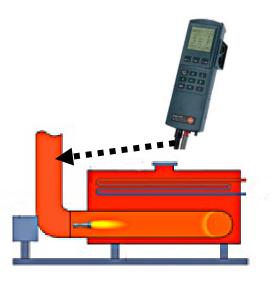


### **Duty Cycle**



# FIRE-TUBE IMMERSION HEATER DESIGN & OPERATION





### **Combustion Analysis**

- 3 T's plus Excess O2

Time – Temperature - Turbulence

+ Excess O2: approx. 3%

Time at Temperature

"NEW" addition of the 4<sup>th</sup> T

- Training!

### Long and Skinny Fire-tube to Improve Heat Transfer

Sept 2006 test heater built, **Wildcat Hills Choke Heater** 6' was added to standard fire-tube Flux = 7,000 Btu/hrft2



- Longer, more slender firetube is not new, many older heaters were built this way and exhibited better efficiency!

Vendor made the fire-tube, shell and process coils longer (with fewer return bends, lowering coil press drop!), shell dia. finished smaller. Fabricated cost of steel ended up similar to standard design.



### **PCOG Fire-tube Heater Field Audits**



- Petro-Canada is actively participating in 10 applications pursuing fire-tube heater efficiency improvements.
- -Assisted by the NRCAN audit process we are attempting to assess 1/3 of our heater fleet/yr. on an ongoing cycle.

### **NRCAN Industrial Energy Audit Incentive Program**



This incentive is designed to help defray the cost of hiring a professional energy auditor to conduct an on-site <u>audit</u> at an industrial facility.



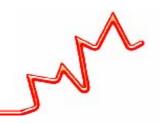


Natural Resources Canada

Ressources naturelles Canada

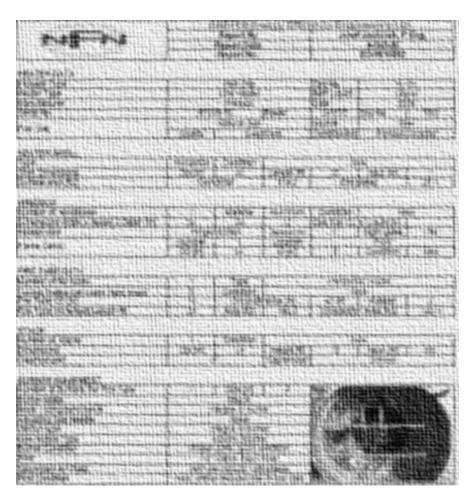


### Fire-tube Heater "Field Audit" Program



		ciency Engineering Ltd. Leduc, AB, T9E 5X3, Canada
NEN	Alberta TEL: (780) 940-3464	Contact: Jozef Jachniak, P.Eng.
	BC TEL: (604) 808-1974	Jachniak@enefen.com
	FAX: 1-866-583-0520	www.enefen.com
<b>EFFICIENCY</b>	RAM M IMMERSION FIRE	-TUBE HEATER
PERFO	DRMANCE EVALUATION	REPORT
report is private and confidential and copy	ing, forwarding or other dissemination or distribut explicit permission from ENEPEN is prohibited	
Daniel Ive	Daniel Date:	7.00
Report by: Jozef Jachniak, P.Eng.	Report Date: 4-Oct-06	Report No.: EG-0610002
Jozef Jachniak, P.Eng.  CUSTOMER DATA  Client: Plant: Area: LSD:	4-Oct-06  Petro Canada Oli and Gas Wildost Hills Gas Plant Wildost Hills Gas Plant OS-13-26-07-W5	Report No.:
Jozef Jachniak, P.Eng.  CUSTOMER DATA  Client: Plant: Area: LSD: Contact:	4-Oct-06  Petro Canada Oli and Gas Wildox Hills Gas Plant Viking Field 05-13-28-07-W5 Phit Croteau	Report No.:
Report by: Jozef Jachniak, P.Eng.  CUSTOMER DATA  Client: Plant: Area: LSD: Contact: Phone Number: Dale:	4-Oct-06  Petro Canada Oli and Gas Wildost Hills Gas Plant Wildost Hills Gas Plant OS-13-26-07-W5	Report No.:





- Stack temp and fuel gas pressure to burner orifice are key variables!

## Combustion Analysis – O2, Excess Air, CO, NOx, comb. efficiency, ambient, bath and stack temp,

S. 435	Report Date	Jazer Jacobnick P. Eng.			Report By	
	Report Date	4-O(-046 2-O(-046)2			Report Date Report No.	
			STACK MEASUREMENT			
ELTRANS COMPOS	1 200 200 200 200	A STATE OF THE PARTY OF THE PAR	Bath Temperature	deg #	95,00	
wing sensor	200	AND DESCRIPTION OF STREET	Ambiert Temperature	deg C	35.00 45.40	
N. IN. P. TOP	448-4-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-	THE PERSON NAMED IN COLUMN TWO IS NOT THE OWNER.	Printed temperate	ded C	8.00	
「中央の大学を表現していませんから、このでは、これでは、これできないない。	大学 できたいとうという 日本 行動 対象をあるとうとということ	THE PROPERTY OF THE PARTY OF TH	Stack Yemperature	deg #	500.00	Stack temperature too high
To be and the first than the second of the second	DE STE SONA 11 P S SONA 15 DE STORA	1		deg C	260 06	
Section 1	The China	Control of the Contro	Oxygen (O2)	1 1	9:40	Stack oxygen high
AND DESIGNATION OF THE PERSON	Tree Secrets	The state of the s	Excess Air (ExA)	ppm	81.0	Excess air high CO within ideal range
CHESTON CONTAINS	THE RESERVE WAS A STREET		Nitrogen Civitie (NCX)	900	40	
roperature cortex vieve	MANAGEMENT AND	THE PROPERTY OF STREET	Arialyzer Measured Efficiency	8	77,80	Efficiency within acceptable range
riper their courts in cole		1460 · 2012 · 2013 · 100	Calculated Efficiency	1	74.58	Based on stack 02, CO, and temperature
新子(A) S(B) 20 (B) 20 (B) (B) (B)	ACM 9 (1031 CCA C0163		Exactive Efficiency	3	82.16	Based on integration of data kit results
TOT THE COURT OF THE PERSON OF THE COURT OF THE CO	100000000000000000000000000000000000000	<b>2000年時期開發開始開發</b>	Achie. Ne Efficiency		85.46	Based on optimized currier size & setup
#3 CABSUP SCENERY	FATAL 521 (\$120 seating, \$100 defice).	<b>医多种种种性性性性性性</b>	DATA KIT HEATEN SERFORMANCE TH	RENDING		
et gas, sitel/ser - 8584	200 (E12) 10 (E12)	<b>建筑铁铁铁铁铁铁铁铁铁铁</b>	1 Data Stubility Check			************
of pressure requires	2000年8月3日 2月1日 1000年8月2日 2月1日	E STATE OF S	# of Samples Collected	Lighten	400152	
e Dar Guerle, 1986. St September 1980/189.		2014年12月1日日本日本日本日本日本日本日本日本日本日本日本日本日本日本日本日本日本日本日	Invalid Temperature Readings		0	
A Safety Strate Traffet	Extract Clark Life 1	<b>建筑建筑建筑设计设计设置</b>	Invario Pressure Readings First Valid Sample Row #		3	111111111111111111111111111111111111111
15 To Williams		<b>建设是"扩展"的。"特别"</b>	Last Valid Sample Row #		12037	
Self Shar rate	14 14	<b>有新疆产业的工作等的现在形</b> 式	# of Valid Contiguous Samples	301134114	12435	
\$64.83.000CEN\$4004	1700	经典型 首把高线面线指线	1 1Cycles			
est 3-compilation	765	per qualità in control de la c	Fue: PT drift threshold	paig	0.1	Pressure transmitter and below which flow-0
<b>州科科 不同一种种一种种的基本之间。在本</b>	नार्वाक्त स्थान करण मात्र । विद्यार (दशकान्य)	to success (week feet) freuen dugge.	Burner Stability Threshold (BST) # of times heater switched ON	D8/0	3.0 36	Fuel pressure below which burner is unslable in excess of 30 seconds.
syllation (0.64 courts showing 454.7.5	CFF SHE HISPAIN FORTH CTT VA	** [1] 是我是一种一个一个一个一个	# of times heater switched OFF			in excess of 30 seconds
			Total number of ON/OFF switches	9.0345.50	78	
IEL FLOW MEXIS WHEMENT	THE PARTY OF THE P		Temperature Control Stability		100%	Percent of cycles > 30 seconds long
ter Burner Ortice Stor it ennem	Fig. 150 Sept 10 are see	Maren a Hoself Dep (Friday) in the	Weaktraid Three		Y 70 750	7
	STITLE OF STREET		Total Sampling Time (Shortest ON Time	nours	50,138	The state of the s
teure - Blocker in	990 1.96 Pilescop 69	gradia acuta deservite	Langest On Time	hours	0.692	
registe - Plantons	4 990 3 4.3 Married 25	THE (400 1-3 (1986) FF	Average ON Time	nours	0.433	
112-11-111-101-101-101-11-11-11	The state of the s		Total Oli Time	hours	15.575	
115	Contract to	Web 4 4th Contributions	Shortest OFF on below BST Time	hours	0.136	The second secon
* Appendix and the second			Langest OFF or below 55T Time Average OFF or below 55T Time	nours	0 987	
or desi Burner Derive	E D. 1 20192 LE grandent o	ette charten justice sette	Total Time below BST	nours nours	0.000	BST+ Burner Stability Threshold
seed Dril 32e	d 3244 (Festle)		Total OFF Time	nours	34.554	The second second
the first of the second section of the section of the second section of the secti	the state of the s	****	DUTY CYCLE = ON / Total Time	18.	31.065	& Tuel pressure above 831 pressure
as dated Coreston	de la	THE PART WHITE	Madgured Temperatures			
nsant Destriction	the second of the second of	West Stateder Applications	Maximum Slack Temperature	deg F	494.32 135.6	
egeneral Plot From egeneral Plot Firing Role	400 STURY C 4	Election in Sutable and the	Average Cooling Time	reinutes		To within 10% of the minimum stack lemoerals

## Heater Utilization – New Equip Performance Validation, in this case heater was only firing 31% duty at < 1/3 design firing rate. Only 10% design utilization, not ideal for a new heater!

と言う	& BHUSTE	Report By Report Date Report No.	6 4-Oct-06	NEN		Report By Report Date Report No.:	etray Engineering U.S. Josef Jacholak, P.Eng. 4-Oct-96 EG-9610002
DATA KIT HEATER PERFORMANCE T	RENDING CO	ird		UPGRADE RECOMMENDATIONS Process improvements	Herms	Urgency of upgrades	3 H high   2 = medium   1 = 6
Measured Foal Flows Total Fuel Consumed During Test	1 50°	24154.6	100000000000000000000000000000000000000	5.1 Target Efficiency (%HHV)	1 85.5	Based on optimized burner	size & selup
Fuer wasted below BST pressure	SOF		BST- Burner Stability Toreshold	1.2 Review Process Requirements	150300	Gross mismatch between f	ring rate a process demand
Del used effectively for heating	305		Based on effective efficiency computation	1.3 Lower Bath Temperature			
Wasted Fuel	1000	0.0%	11.000	1.4 Shut heater of when not needed			
Askinum Fuel Flow	SCFH	452.8	A CHARLES OF THE PARTY OF THE P	5 Review temp, sensor location			
Werage Fuel Flow During ON TIME	SCFH	438.7		1.6 Review external heat losses	- Diameter		
werace Fuel Flow below 85.7	SOFH	1003		1.7 Repeat data located	1 1 1 1 1 1 1		
onstant Equivalent Fuel Flow	SCFH	136.3	1 Tana Salar Park Park Park Park Park Park Park Pa	1.5 Repeat stack analysis		The printed sharing	
onstant Plict Fuel Flow (nominal)	\$CFH.	30.0		1.9 Other process improvement			
onetant Total Fuel Flow	\$CFH	166.3		Maintananna shpnovarrients 2.1 Clean flathe cell			
		Calculus.			L. Carrier	Abilition Constitution	
MERMAL PERFORMANCE	istimato			2.2 Clean fire-table 2.3 For fire-table	il in the later	\$	
current Peak Condition				2 4 Adjust combustion air			
ue Peak input incl. pilot	MMBTUN	0.49	4 1 - 5 T 1 2 - 7 - 7 - 7 - 7 - 7 - 7 - 7 - 7 - 7 -	2.5 Adjust fuel gas pressure	+	<del>-  </del>	****
rocess Feak Heat Input	MMSTLify	0.40		2.5 Clean burner			
ine-Tube Surface Heat Flux Rate	81Uhrt2	2,133	The Harris Head of the Control of th	2.7 Clean plict	100		
rossectional Reat Flux	BTURNIN2	2,825	distributed by the Charles of Salary	2.8 Set up control logo	1000		
SNAMESTE Continuous Flow	Provide to		the title that the time to be a second to the time to be a second to the time to be a second to the time to be	2.9 Test flame arrester			
onstant Equivalent Fuel input	WM BTURY		10 1012 1012 1012 1012 1012 1012 1012 1	2 10 Other maintenance improvement			
Process Heat Regultement	MMBTURY		A Triangular and the Control of the	(XInot Medifications			
Fire-Tube Surface Heat Flux Rate Crossectional Heat Flux	BTU/hrft2 BTU/hrft2		A THE PARTY OF THE	3.1 Add / modify secondary air prate	37(3.1)	Necessary to control excess	s air
Tra-Tube Reting	E TOWNSEL	SUM		3.2 Replace burner	332-833	Improper burner size, arran	igentent or condition
care plate rating (per tube)	[ MM BTUM	1.50	AND THE RESIDENCE OF THE PARTY	3.3 Replace plict	문장말(함)		
riginal tating based on 10K heat flux	MM BTURY		Hepter designed for 7000 87U hr/ft2 heat flux	3.4 Align burner			
Attization of Original Rating	1 3	6.8%	19.7% based on 7000 87 Uhriff2 heat flux	3.5 Aligh plot	10000000		
	1	100		3.6 Change burner office			
URNER SIZING ANALYSIS	3-47-78-78-78			3.7 Ramove dume: adjusting needle. 3.8 Add view bort	h china haba	Animitimizations.	Thirting the state of the state of
Willer Capton des Mainten (1975)		THE SECTION	CONTRACTOR OF THE PROPERTY OF	3 9 Replace fiame cell		A STATE OF THE PARTY OF THE PAR	
xira 8umer Capacity				3 10 Install burner union			******************
otal Required Fuel Input	MM-5TUN	0.21	And the second section and a second s	3 11 Install housing couplings;		•	
esé plot capacity	MINISTURY			3. 12 Install wiring seal on windbox	********		
fain Burner Puet Input	MWSTURE	0.18		3.13 Change requiator spring	THE PERSON NAMED IN		
Soleting Burnerie	The latest and the la	35, 3, 11, 13, 12,		3 14 Change regulator orfice	-47000		
uriber of burners per fire-tube	1	S - 11 - 1		3.15 Replace pressure gauges	1 1 1 1 1 1 1	Note the Property of the Parish	
xisting Burner Size	36	4 100		3.16 Change TSH TSHIR to shap acting:			
tevised Buther \$190g - Single numer	eelectron	uzinta		3.17 Other minor modification	a Cartill	product products	
ine-Tube Size	in	14		Major Modifications	1111	A STATE OF THE PROPERTY OF THE PARTY OF THE	
Aaximuni allowable burner size	0	4	Limited by fire-tube diameter	4.1 (west/replace 1979, spetter flow detection	S FOR (SI)		
lurner size to meet capacity	in	1.5	Burner size less than maximum	4.2 Install power source	LE MILL	A STREET WATER STREET	This is a second to the second
Askimum nominal burner capacity	STURM	212,000		4.3 Modify to interruptible pilot	HICKORY.	Charles of the same	al-remainment of the
finithum nominal dumer capacity	BTUM	53,000	Control of the Contro	4 & April replace safety shutoff valve(s)	1 517 177		
Maximum Burner Fire	100	83.90		4.5 Add / replace mariual valve(s)		A commission of the commission	The Address of the London
localinal mandetum burner pressure	paig	20.00		4.6 Add replace LSLU			
Onfice size required	10	0.0746 843		4.7 Add / replace TSH / TSHH	14	- FEMALES SERVICES	THE PARTY OF THE PARTY OF THE PARTY.
di Diameter	10	0.0760		4.8 Add fluel gas PSHH 1 PSLL 4.9 Address other safety Issue	distant.	- I I I I I I I I I I I I I I I I I I I	

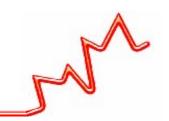
Opportunity to save capital building heaters smarter, smaller!

## **Summary Sheet of Expected Savings**

NEN		Report By:	rergy Efficiency Engineering Ltd. Jozef Jackniak, P. Eng.
	Report Do Report N		4-Oct-06 EG-0610062
UPGRADE RECOMMENDATIONS	E THE	Lirgency	of upgrades 1 5 + high 2 + menium 1 +
Water Word treations (danied)		STORY OF	
4.10 Add HULC/OFF control		Barrier Contract	
4 (1) Axid modulating control	Day to be to be		
f. 12 Gograde venting system			
4.13 Change pressure regulator(s)			
4 14 Modify Instrument gas system			
4 15 Replace currier housing	201121111		
4.16 insurate heater		erennen e	
4.17 insulate stack	40000		
d 18 Add slack height	2011/2/12	拉耳尼科斯克	
4.19 Add stack thermometer			
4.20 Replace fire-fube	241417011	District	
4.21 Add lighting / view part to fire tube		TRUBBLE	
4:22 Other major modification	STATE		
Measured Constant Total Fuel Flow Seasonal Fuel Flow Correction	\$2F4	165.3 25.00	To compensate for whiter demand
	SCF	1,820,698	Blased on 6760 hrs/A operation
Annual Fuel Consumption	4 7 190	\$1,563	constant total fuel flow
(hished on houseast 1000 SETU / SCF 1997)	MAKETU	1321	
	GJ	1921	
Effective Efficiency	5	82.16	
		98.00	
Achievable efficiency	- 54	85,46	
Achievable efficiency improvement in efficiency		85,46 3,30	
	5 %	85,46 3,30 3,96	
more/enert in efficiency	SOF	85,46 3,30 3,96 70,276	
ingrovement in efficiency Annual Fuel Savings Due to Efficiency	SOF TO	85,46 3,30 3,96	
ingrovement in efficiency Annual Fuel Savings Due to Efficiency	N SOF m3 MM/STU	85.46 3.30 3.96 70.276 1.990 70	
improvement in efficiency Annual Fuel Savings Due to Efficiency	SOF m3 MV STU GJ	85,46 3,86 3,86 70,276 1,990 75 74	
ingrovement in efficiency Annual Fuel Savings Due to Efficiency Improvement	N N N SOF m3 MV 6TU GV 3CF	85.46 3.30 3.96 70.276 1.990 70	
ingrovement in efficiency  Annual Fuel Savings Due to Efficiency improvement  Annual Fuel Savings Due to Elimination of	N N N SOF m3 MV 8TU GU GU SOF N3	85,46 3,36 3,96 70,276 1,990 70 74 164 5	
ingrovement in efficiency  Annual Fuel Savings Due to Efficiency improvement  Annual Fuel Savings Due to Elimination of	SOF MV STU GJ MV STU GJ MM STU	85,46 3,30 3,96 70,276 1,990 70 74 1,64 5 0	
ingrovement in efficiency  Annual Fuel Savings Due to Efficiency improvement  Annual Fuel Savings Due to Elimination of	SOF MYSTU GU SOF MID MM BTU	85,46 3,30 3,96 70,276 1,990 70 74 1,64 5 5 0	
ingrovement in efficiency  Annual Fuel Savings Due to Efficiency improvement  Annual Fuel Savings Due to Elimination of	SOF MAYERU SOF MAYERU SOF MAYERU GOS	85,46 3,30 3,36 70,276 1,990 70 74 164 5 0 70,440	
ingrovement in efficiency  Annual Fuel Savings Due to Efficiency improvement  Annual Fuel Savings Due to Elimination of Fuel Pressures below BST	S SC	85,46 3,30 3,86 70,276 1,990 70 74 1,64 5 0 70,440 1,995	
ingrovement in efficiency  Annual Fuel Savings Due to Efficiency improvement  Annual Fuel Savings Due to Elimination of Fuel Pressures below BST	SOF m3 MV eTU G3 SOF F3 MM ETU G3 SOF F3 MX eTU	85,46 3,30 3,96 70,276 1,990 70 74 1,64 5 0 9 70,440 1,955	
	S SC	85,46 3,30 3,86 70,276 1,990 70 74 1,64 5 0 70,440 1,995	



### Conclusion



### The means to achieve improved efficiency is as simple as:

- training theory, operations, combustion testing with analyzer,
   CMMS (EMPAC)
- manage excess air in combustion
- manage the burner duty cycle
- strive for 82% combustion efficiency (depends on service, i.e. bath approach temp)
- provide adequate insulation to reduce energy demand (reduction is a 100% improvement)
- steward regular combustion analysis and inspection of heaters spring and fall, focusing on duty cycles, CO in combustion, excess air and stack temperature (fire-tube exit temp)
- integrate burner duty with process demand where possible
- design new equipment to address the items above (burners and firetubes)
- maintain CMMS records of fired equipment
- DESIGN YOUR HEATER TO MEET THE SERVICE DUTY, FIRING MODE, ENGAGE (OME), PRODUCTION AND PROJECT GROUPS!